

THE CHINESE UNIVERSITY OF HONG KONG DEPT OF MECHANICAL & AUTOMATION ENG



ENGG5403 Linear System Theory & Design

Assignment #6

by

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Problem 1

Write the system to be controlled in Homework Assignment 5 in the following form

$$\Sigma: \begin{cases} \dot{x} = A \ x + B \ u + E \ \tilde{w} \\ y = C_1 \ x + D_1 \ \tilde{w} \\ z = C_2 \ x + D_2 \ u \end{cases}$$

with

 $\Sigma : \begin{cases} \dot{x} = A \ x + B \ u + E \ \tilde{w} \\ y = C_1 \ x + D_1 \ \tilde{w} \\ z = C_2 \ x + D_2 \ u \end{cases}$ $\tilde{w} = \begin{pmatrix} v(t) \\ w(t) \end{pmatrix}, \text{ the combination of the input and measurement noises.}$

- 1. Determine the best achievable H_{∞} -norm of the closed-loop system from \tilde{w} to z?
- 2. Design an H_{∞} suboptimal control law such that the H_{∞} -norm of the resulting closedloop system is reasonably close to the optimal value.
- 3. Plot the singular value of the closed-loop system and find its H_{∞} -norm.
- 4. Find the resulting gain and phase margins of the system under the control law.
- 5. Assume that there is an unstructured but stable perturbation, Δ , presented in the given plant. Give the range of $\|\Delta\|_{\infty}$ so that the closed-loop would remain stable.

Solution:

$$\begin{cases}
\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -1 & -1 & 1 & 1 \\ 0 & 0 & 0 & 1 \\ 1 & 1 & -1 & -1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \\ 0 \\ 0 \end{bmatrix} u + \begin{bmatrix} 0 & 0 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} v(t) \\ w_1(t) \\ w_2(t) \end{bmatrix}$$

$$y = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} v(t) \\ w_1(t) \\ w_2(t) \end{bmatrix}$$

$$z = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

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$$x_1 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_2 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_3 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_4 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

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$$x_5 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_6 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_7 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$x_8 = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

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- 1. Try gm8s_sc for many times, we find $\gamma_{\infty}^* = 0.75259$ is the best achievable H_{∞} norm of the closed-loop system from \tilde{w} to z.
- 2. When designing the H_{∞} optimal controller, we use $\gamma = 0.805$. Choose $\epsilon = 0.01$ and solve h8care function in MATLAB to calculate P and Q in following equation

$$A^T P + PA + \tilde{C}_2^T \tilde{C}_2 + \gamma^{-2} P \tilde{E} \tilde{E}^T P - \left(PB + \tilde{C}_2^T \tilde{D}_2 \right) \left(\tilde{D}_2^T \tilde{D}_2 \right)^{-1} \left(\tilde{D}_2^T \tilde{C}_2 + B^T P \right) = 0 \quad (2)$$

and

$$QA^T + AQ + \tilde{E}\tilde{E}^T + \gamma^{-2}Q\tilde{C}_2^T\tilde{C}_2Q - \left(Q\tilde{C}_1^T + \tilde{E}\tilde{D}_1^T\right)\left(\tilde{D}_1\tilde{D}_1^T\right)^{-1}\left(\tilde{D}_1\tilde{E}^T + \tilde{C}_1Q\right) = 0 \quad (3)$$

The results of P and Q are

$$P = \begin{bmatrix} 0.1420 & 0.0099 & 0.0001 & -0.0001 \\ 0.0099 & 0.0013 & 0.0001 & 0.0001 \\ 0.0001 & 0.0001 & 0.0002 & 0.0001 \\ -0.0001 & 0.0001 & 0.0001 & 0.0002 \end{bmatrix}$$
(4)

and

$$Q = \begin{bmatrix} 3.4975 & 1.4166 & 3.0796 & 1.6504 \\ 1.4166 & 0.9522 & 1.1241 & 0.8309 \\ 3.0796 & 1.1241 & 2.8024 & 1.3486 \\ 1.6504 & 0.8309 & 1.3486 & 0.9637 \end{bmatrix}$$
 (5)

F and K can be gotten from following equations

$$F = -\left(\tilde{D}_2^T \tilde{D}_2\right)^{-1} \left(\tilde{D}_2^T \tilde{C}_2 + B^T P\right) = \begin{bmatrix} -69.7284 & -10.9821 & -1.0077 & -1.0377 \end{bmatrix}$$
(6)

and

$$K = -\left(Q\tilde{C}_{1}^{T} + \tilde{E}\tilde{D}_{1}^{T}\right)\left(\tilde{D}_{1}\tilde{D}_{1}^{T}\right)^{-1} = \begin{bmatrix} -3.4999 & -3.0815\\ -1.4172 & -1.1246\\ -3.0815 & -2.8040\\ -1.6513 & -1.3494 \end{bmatrix}$$
(7)

Also, the eigenvalues of the closed-loop system are verified

$$\lambda = \begin{cases}
-369.08 \\
-5.99 + 5.91i \\
-5.99 - 5.91i \\
-1.03 + 1.02i \\
-1.03 - 1.02i \\
-0.50 + 0.87i \\
-0.50 - 0.87i \\
-0.47
\end{cases} \tag{8}$$

which are all in the left-half plane. Therefore, the H_{∞} -suboptimal output feedback law is then given by

$$\begin{cases} \dot{x}_{cmp} = \begin{bmatrix} -208.7362 & 1.0000 & -183.1922 & -0.0000 \\ -157.0601 & -11.9787 & -75.6699 & -0.0374 \\ -183.1922 & 0.0000 & -160.8651 & 1.0000 \\ -98.2744 & 1.0000 & -88.0212 & -1.0000 \end{bmatrix} x_{cmp} + \begin{bmatrix} 208.7362 & 183.1922 \\ 86.3534 & 75.6626 \\ 183.1922 & 160.8651 \\ 99.2744 & 87.0212 \end{bmatrix} y \\ u = \begin{bmatrix} -69.7282 & -10.9820 & -1.0077 & -1.0377 \end{bmatrix} x_{cmp} \end{cases}$$
(9)

3. Using Matlab, the singular value of the closed-loop system can be plotted as shown in Figure 1.

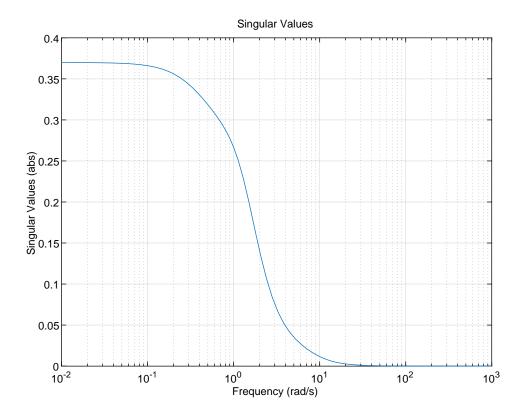


Figure 1: singular value of the closed-loop system.

And the H_{∞} norm of the system is equal to 0.7264.

- 4. The resulting gain and phase margins of the system under the control law are all equal to ∞ .
- 5. The range of the unstructured but stable perturbation is

$$\|\Delta\|_{\infty} < \frac{1}{\gamma} = 1.3766 \tag{10}$$

The codes for this question are listed below:

```
%% Q6-1
   clc; clf; clear all; close all;
   A = [
3
4
        0 1 0 0;
5
        -1 -1 1 1;
        0 0 0 1;
6
7
        1 1 -1 -1;
8
        ];
9
10
```

```
11
        1;
12
        0;
13
        0;
14
        ];
15 \quad C1 = [1 \ 0 \ 0 \ 0;
        0 0 1 0;];
16
17 \quad C2 = [1 \quad 0 \quad 0 \quad 0];
18 D1 = [0 1 0;
19
        0 0 1;];
20 D2 = 0;
21 \quad E = [0 \ 0 \ 0;
22
        1 0 0;
23
         0 0 0;
24
         0 0 0;];
25 epsilon = 0.01;
26 % C2 = [C2; epsilon*eye(size(C2,2)); zeros(size(D2,2),size(C2,2));];
27 % D2 = [D2; zeros(size(C2,2),size(D2,2)); epsilon*eye(size(D2,2));];
28 % E = [E epsilon*eye(size(E,1)) zeros(size(E,1),size(D1,1))];
29 % gms8 = gm8star(A,B,C2,D2,E);
30 \text{ gamma} = 0.805;
31 % gm8s_sc(A,B,E,C1,D1,C2,D2,gamma);
32 C2 = [C2; epsilon*eye(size(C2,2)); zeros(size(D2,2),size(C2,2));];
33 D2 = [D2; zeros(size(C2,2),size(D2,2)); epsilon*eye(size(D2,2));];
34 E = [E \text{ epsilon} * eye(size(E,1)) \text{ zeros}(size(E,1), size(D1,1))];
35 D1 = [D1 zeros(size(D1,1), size(E,1)) epsilon*eye(size(D1,1))];
36 \% \text{ gms8} = \text{gm8star}(A, B, C2, D2, E);
37 P = h8care(A, B, C2, D2, E, gamma);
38 Q = h8care(A',C1',E',D1',C2',gamma);
39 % % F = -((D2'*D2)^{-1})*(D2'*C2+B'*P);
40 % % K = -(Q*C1'+E*D1')*((D1*D1')^-1);
41 % % epsilon = 0;
42 [F,K,Acmp,Bcmp,Ccmp,Dcmp,EigCL] = h8out(A,B,E,C1,D1,C2,D2,gamma,epsilon);
43 % % EigCL
44 % % gm8 = sqrt(max(eig(P*Q)))
45 \quad C1 = [1 \quad 0 \quad 0 \quad 0;
46
        0 0 1 0;];
47 \quad C2 = [1 \quad 0 \quad 0 \quad 0];
48 D1 = [0 1 0;
49
        0 0 1;];
50 D2 = 0;
51 \quad E = [0 \ 0 \ 0;
52
        1 0 0;
53
         0 0 0;
54
         0 0 0;];
55 Acl = [A+B*Dcmp*C1 B*Ccmp; Bcmp*C1 Acmp];
56 Bcl = [E+B*Dcmp*D1; Bcmp*D1];
```

```
57 Ccl = [C2+D2*Dcmp*C1 D2*Ccmp];
58 Dcl = D2*Dcmp*D1;
59 \quad [num, den] = ss2tf(Acl, Bcl, Ccl, Dcl, 1);
60 \text{ sys} = tf(num, den);
61 p= sigmaoptions;
62 p. MagUnits='abs';
63 fig1 = figure(1);
64 sigma(sys,p);
65 grid on;
66 % a = get(gca,'XTickLabel');
67 % set(gca,'XTickLabel',a,'FontName','Times','fontsize',12);
68 set(gcf,'renderer','painters');
69 filename = "Q6_SV"+".pdf";
70 saveas(gcf, filename);
71 close(fig1);
72 [Gm, Pm, Wcg, Wcp] = margin(sys);
```

Problem 2

Consider a linear time-invariant system characterized by

$$\Sigma : \begin{cases} \dot{x} = Ax + Bu + Ew \\ z = C_2 x + D_2 u \end{cases}$$
 (11)

where $C_2 = 0_{m \times n}$, $D_2 = I_m$, and where $x \in \mathbb{R}^n$, $u \in \mathbb{R}^m$, $w \in \mathbb{R}^l$ and $z \in \mathbb{R}^m$, are the state, control input, disturbance input and controlled output, respectively. Assume that the state variable x is available for feedback, i.e, the measurement output y = x, and assume that (A, B) is stabilizable and (A, B, C_2, D_2) has no invariant zeros on the imaginary axis.

- (a) Show that the subsystem (A, B, C_2, D_2) has a total of n invariant zeros and are given by $\lambda(A)$, i.e., the eigenvalues of A.
- (b) Show that there exist an $n \times n$ nonsingular transformation T such that

$$\tilde{A} = T^{-1}AT = \begin{bmatrix} A_- & 0\\ 0 & A_+ \end{bmatrix} \tag{12}$$

where A_{-} and A_{+} are stable and unstable matrices, respectively.

(c) Let us define a state transformation $x = T\tilde{x}$, where T as given in Part (b). It is easy to verify that the given system Σ can be transformed into the following:

$$\begin{cases} \dot{\tilde{x}} = \begin{bmatrix} A_{-} & 0 \\ 0 & A_{+} \end{bmatrix} \tilde{x} + \begin{bmatrix} B_{-} \\ B_{+} \end{bmatrix} u + \begin{bmatrix} E_{-} \\ E_{+} \end{bmatrix} w \\ z = \begin{bmatrix} 0 & 0 \end{bmatrix} \tilde{x} + Iu \end{cases}$$
(13)

where B_- , B_+ , E_- , and E_+ are respectively appropriate constant matrices. Show that (A, B) is stabilizable if and only if (A_+, B_+) is controllable.

(d) Show that the solution to the corresponding H_2 Riccati equation for the transformed system in Part (c), if existent, can be partitioned as follows

$$P = \begin{bmatrix} 0 & 0 \\ 0 & P_+ \end{bmatrix}, P_+ > 0 \tag{14}$$

Find the H_2 optimal state feedback control law $u = F\tilde{x}$ for the transformed system in terms of P_+ . Show that the resulting closed-loop system has poles at λ (A_-) and λ $(-A_+)$.

(e) Show that $\gamma_2^* = 0$, i.e., the disturbance can be totally rejected from the controlled output, if and only if $E_+ = 0$, i.e., the disturbance is not allowed to enter the unstable invariant zero subspace.

Solution:

(a) or a linear time-invariant system, the invariant zeros are defined as the values of *s* for which the system has no response to the input. We can find the invariant zeros by computing the transfer function from *u* to *z* and looking for the zeros of the determinant of the system matrix. In our case, the transfer function is given by:

$$G(s) = C_2(sI - A)^{-1}B + D_2 = I_m.$$
(15)

Since C_2 is a zero matrix and D_2 is the identity matrix, the transfer function is always equal to I_m and does not depend on the system's poles or zeros. However, we can still find the invariant zeros by computing the transfer function from w to z. In this case, the transfer function is given by:

$$H(s) = C_2(sI - A)^{-1}E.$$
 (16)

To find the invariant zeros, we want to find the values of s for which the determinant of (sI - A) is zero:

$$\det(sI - A) = 0. \tag{17}$$

This equation gives us the eigenvalues of A, which are the invariant zeros of the subsystem (A, B, C_2, D_2) . Thus, the subsystem has a total of n invariant zeros, given by the eigenvalues of A.

(b) Since (A, B) is stabilizable, there exists a nonsingular transformation T such that the transformed A matrix, \tilde{A} , is in the block upper-triangular form with stable and unstable parts A_- and A_+ , respectively. This transformation is obtained through modal decomposition. To show this, we first find the eigenvectors of A corresponding to its eigenvalues. Let v_i be the eigenvector corresponding to the eigenvalue λ_i of A:

$$Av_i = \lambda_i v_i. \tag{18}$$

We can then construct the transformation matrix T by placing the eigenvectors as its columns:

$$T = \begin{bmatrix} | & & | \\ v_1 & \cdots & v_n \\ | & & | \end{bmatrix}. \tag{19}$$

Applying this transformation, we obtain the block upper-triangular form for \tilde{A} :

$$\tilde{A} = T^{-1}AT = \begin{bmatrix} A_- & 0\\ 0 & A_+ \end{bmatrix},\tag{20}$$

where A_{-} and A_{+} are stable and unstable matrices, respectively.

(c) We have already defined the state transformation $x = T\tilde{x}$. Let's rewrite the system Σ in terms of \tilde{x} :

$$\dot{\tilde{x}} = T^{-1}\dot{x} = T^{-1}(Ax + Bu + Ew) = T^{-1}A(T\tilde{x}) + T^{-1}Bu + T^{-1}Ew.$$
 (21)

Using the definition of \tilde{A} and noting that $T^{-1}A = \tilde{A}T^{-1}$, we can write

$$\dot{\tilde{x}} = \tilde{A}\tilde{x} + T^{-1}Bu + T^{-1}Ew. \tag{22}$$

Now, let's define $\tilde{B} = T^{-1}B$ and $\tilde{E} = T^{-1}E$. Then, we can partition these matrices as:

$$\tilde{B} = \begin{bmatrix} B_- \\ B_+ \end{bmatrix}, \ \tilde{E} = \begin{bmatrix} E_- \\ E_+ \end{bmatrix}.$$
 (23)

With these definitions, the transformed system takes the form:

$$\begin{cases}
\dot{\tilde{x}} = \begin{bmatrix} A_{-} & 0 \\ 0 & A_{+} \end{bmatrix} \tilde{x} + \begin{bmatrix} B_{-} \\ B_{+} \end{bmatrix} u + \begin{bmatrix} E_{-} \\ E_{+} \end{bmatrix} w \\
z = \begin{bmatrix} 0 & 0 \end{bmatrix} \tilde{x} + Iu
\end{cases} (24)$$

The pair (A, B) is stabilizable if and only if the unstable part of the system, (A_+, B_+) , is controllable. This is because stabilizability implies that we can place the closed-loop poles in the left-half plane, and this can only be achieved if we can control the unstable part of the system.

(d) The H_2 optimal control problem is related to minimizing the H_2 norm of the transfer function from w to z. The optimal control law can be found by solving the H_2 Riccati equation:

$$A^T P + PA - PBB^T P + Q = 0, (25)$$

where $Q = C_2^T C_2$ and $R = D_2^T D_2$. In our case, $C_2 = 0_{m \times n}$ and $D_2 = I_m$, so $Q = 0_{n \times n}$ and $R = I_m$. Now, if the solution to the Riccati equation exists, we can partition the matrix P as:

$$P = \begin{bmatrix} 0 & 0 \\ 0 & P_+ \end{bmatrix},\tag{26}$$

with $P_+ > 0$. The H_2 optimal state feedback control law is given by $u = F\tilde{x}$, where $F = -(R + B^T P B)^{-1} B^T P A$. Using the partitioned form of P, the feedback control law becomes:

$$u = -(I + B_{+}^{T} P_{+} B_{+})^{-1} B_{+}^{T} P_{+} A_{+} \tilde{x}_{+}. \tag{27}$$

The resulting closed-loop system has the state-space representation:

$$\dot{\tilde{x}} = \begin{bmatrix} A_{-} & 0 \\ 0 & A_{+} + B_{+}F \end{bmatrix} \tilde{x} + \begin{bmatrix} E_{-} \\ E_{+} \end{bmatrix} w. \tag{28}$$

The poles of the closed-loop system are given by the eigenvalues of the matrix in the above equation. Since A_- is stable and the feedback law stabilizes the unstable part $A_+ + B_+ F$, the closed-loop poles are given by $\lambda(A_-)$ and $\lambda(-A_+)$.

(e) The disturbance can be totally rejected from the controlled output, i.e., $\gamma_2^* = 0$, if and only if the transfer function from w to z is identically zero. This can be achieved if and only if $E_+ = 0$, meaning the disturbance is not allowed to enter the unstable invariant zero subspace. If $E_+ = 0$, then the disturbance only affects the stable part of the system, and the feedback control law can completely reject the disturbance from the controlled output.