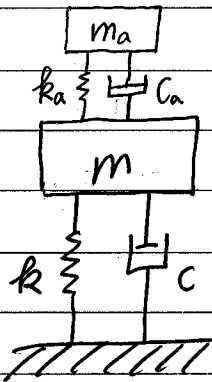


General Structural System (Lumped - Parameter):

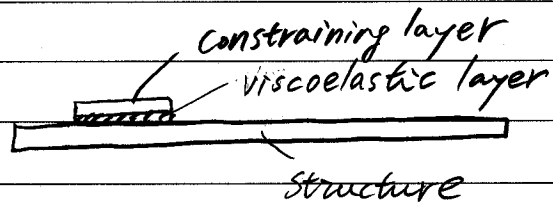
$$M\ddot{X} + C\dot{X} + KX = F(t)$$

- Passive Control

- parameters (M, c, K) are synthesized off-line
- stable, no power requirement
- fixed design, no feedback
- examples



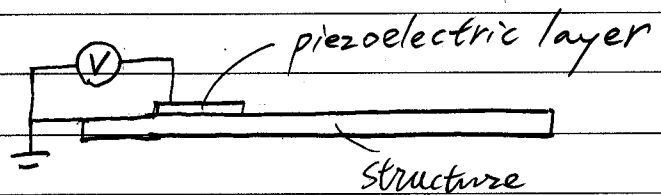
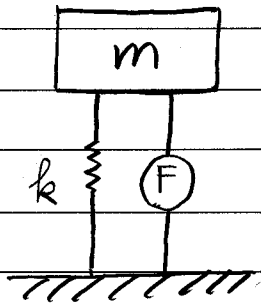
Vibration Absorber



(Passive) Constrained Layer
Damping Treatment

- Active Control

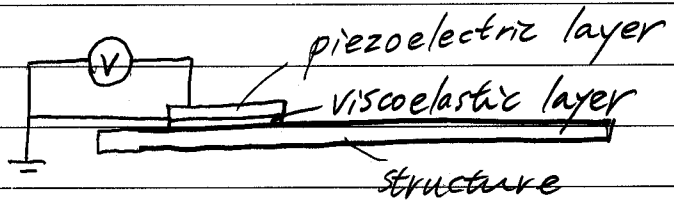
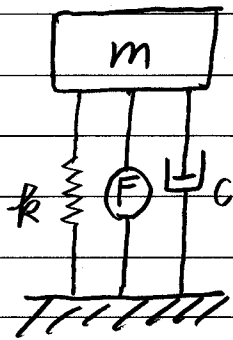
- force actuators are used
- feedback, high performance
- high power requirement, potential for instability
- examples



- Active - Passive Control

- integrate active systems with well-designed passive elements
- combine the advantages of both active (feedback, high performance) and passive (stable, fail-safe) systems
- low power requirement

- examples

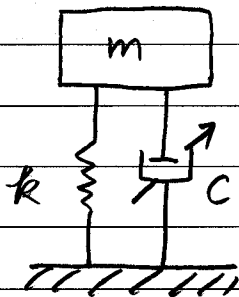


Active Constrained Layer (ACL)
Damping Treatment

• Semi-Active Control

- parameters (M, c, k) are controllable

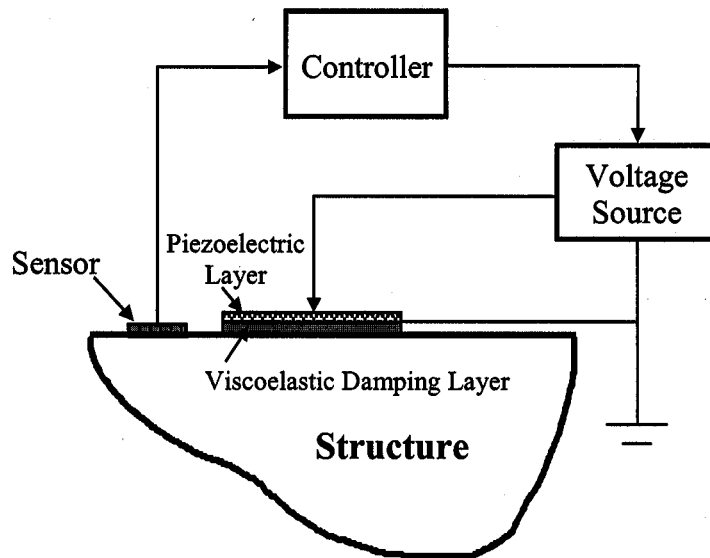
e.g. adaptable energy dissipation devices



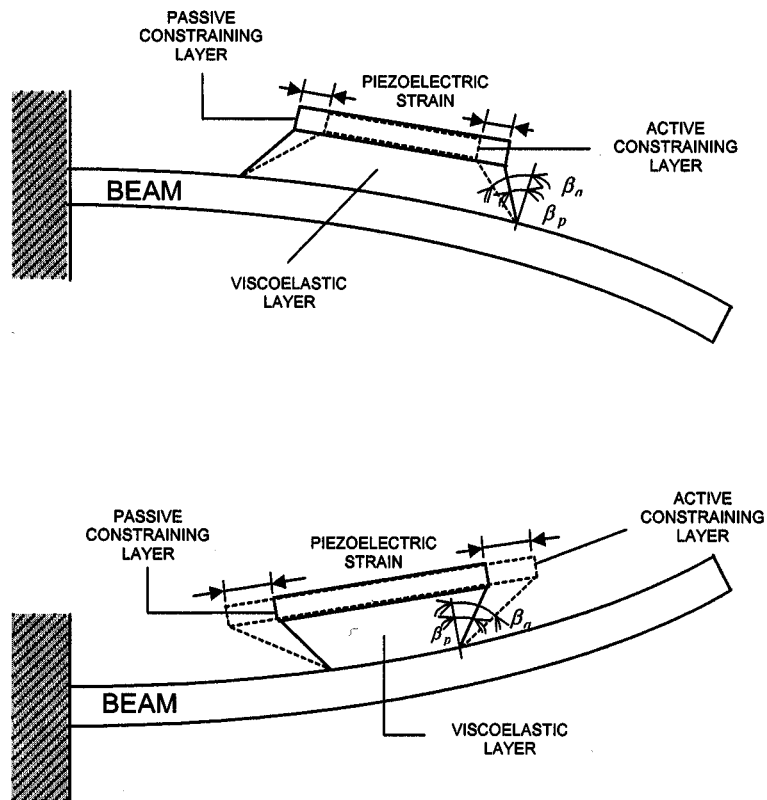
controllable damper
such as MR damper

- also have the advantages of both active (feedback, high performance) and passive (stable, fail-safe) systems

- low power requirement



Structure with the active constrained layer (ACL) damping treatment



Operating principle of the ACL treatment

$$M\ddot{q} + C\dot{q} + Kq = f(t) \quad (a)$$

where M, C, K are mass, damping, and stiffness matrices ($N \times N$)

q is vector of generalized coordinates ($N \times 1$)

f is vector of generalized force ($N \times 1$)

Introducing the state vector

$$X(t) = \begin{Bmatrix} q(t) \\ \vdots \\ \dot{q}(t) \end{Bmatrix} \quad (b)$$

We can rewrite eq. (a) in state space form:

$$\dot{X} = AX + BU$$

$$Y = C_0 X + DU \quad (c)$$

where

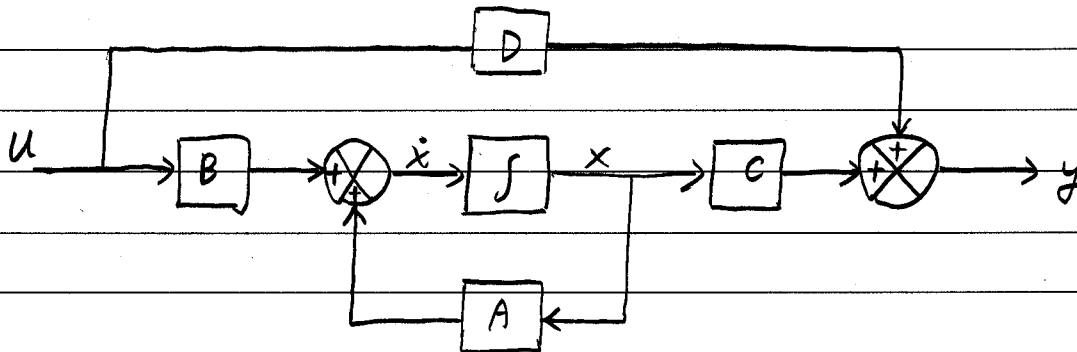
$$A = \begin{bmatrix} 0 & I \\ -M^{-1}K & -M^{-1}C \end{bmatrix} \quad (d)$$

$$B = \begin{bmatrix} 0 \\ M^{-1} \end{bmatrix} \quad \text{for } u = f(t)$$

$$\dot{x}(t) = Ax(t) + Bu(t)$$

(*)

$$y(t) = Cx(t) + Du(t)$$



Block Diagram